

# Numerical Study of a Cracked Pipe Before And After Using Two Different Repair Methods

Abderraouf Khireche\*, Tidjani Ahmed Zitouni\* and Zohra Labeled\*\*

**Keywords :** pipeline, fracture, repair, pressure.

## ABSTRACT

Pipelines are the cheapest and safest way to transfer fluids in industry. The operation of pipelines, on the other hand, raises the risk of failures such as internal and exterior corrosion, cracking, and warping. It should be noted that non-destructive testing guarantees the correct operation of pipelines before and after failures. As a result, when damage occurs, it is important to fix it in order to avoid loss of life and material. In our study, we generated an external fracture in a pipeline by using the API X80M as the construction material. We then performed a comparative numerical analysis using the ANSYS software, simulating two methods of repair, the first using a half-shell and the second using a composite patch (epoxy carbon). The Von Mises stresses and stress intensity variables were really the parameters considered in our investigation.

## INTRODUCTION

The prediction of fractures, as well as the durability and safety of pipelines, are essential in many practical transportation applications, such as hydrocarbon distribution (Miller, 1988; Kim, 2003). The erosion or complete decomposition of these pipelines over time, depending on the corrosive media, will result in very significant economic losses (Arav, 1997). Fracturing, which may take various forms, is one of the most hazardous issues. Depending on their direction, cracks can be classified as longitudinal or transverse (Mahdi et Hasnaoui, 2018). It is well

*Paper Received November, 2021. Revised April, 2022. Accepted June, 2022. Author for Correspondence: Abderraouf Khireche.*

\* PhD student, Department of Mechanical Engineering, Faculty of Technical Sciences, Mentouri Brothers University, Constantine 1, Algeria.

\* PhD student, Department of Mechanical Engineering, Faculty of Technical Sciences, Mentouri Brothers University, Constantine 1, Algeria.

\*\* Professor, Department of Mechanical Engineering, Faculty of Technical Sciences, Mentouri Brothers University, Constantine 1, Algeria.

known in fracture mechanics that the development of a crack is mostly determined by the stress state and the local strain near the crack tip. When the effects of plasticity are significant, the modeling of fracture propagation must take into consideration the material's actual behavior. Any particle entry into significant fractures under cyclic loading might cause pipeline damage and burst (Brekke, 1986).

There is always a risk of pipeline damage due to different situations such as corrosion, structural defects, and ground movement (Zedadma et Khallaf, 2018; Khireche et Labeled 2018). There are numerous sorts of pipeline repairs that may be performed to extend their life. In this case, we employed two distinct repair procedures (half-shell welding and composite patch repair), the objective of which is to decrease or eliminate the fracture in terms of resistance and service life of the pipe. Our task is to find the most effective repair procedure possible in order to avoid any damage or rupture of the fractured pipe. We have three elements: The first step was to determine the Von Mises stresses of an ideal tube as the internal pressure is varied. The second portion displays the results of a cracked tube in order to calculate its resistance limit based on the stress intensity factor. To avoid any damage or rupture, we repaired, either with a composite patch or a half-shell. In the third section, we provide the Von Mises stresses and stress intensity factor findings, which will help us to choose the optimum repair strategy.

## UNCRACKED PIPE

We consider a pipeline made of **API X80M** material to be linearly elastic, with the following mechanical properties:

Table1. Mechanical properties of ISO 3183 X80M steel (Santos, Hermenegildo, Afonso, Marinho, Paes et Ramirez, 2010)

Properties	MPa	Ksi
Yield strength (YS)	568	82
Tensile strength (TS)	686	100
YS/TS (%)	83	
% Elongation	44	
Average micro-hardness (HV0.2)	235	

**Geometry the tube**

On a pipeline with an internal diameter of 722 mm, a thickness of 10 mm, and a length of 1000 mm, we applied four different pressures (5, 6, 7, and 8 MPa). We a test of "grid-independence" to find a proper mesh size, as shown in Fig. 1

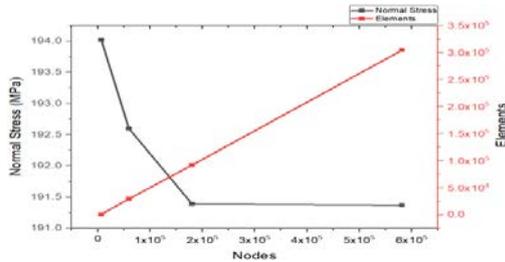


Fig. 1 The grid-independence

- Stress results line become horizontal that show mesh independence.
- Allowable change under 1 %.
- Increasing in elements number according to normal stress and nodes number.

The following graphs illustrate the results of the ideal pipe pipeline simulation:

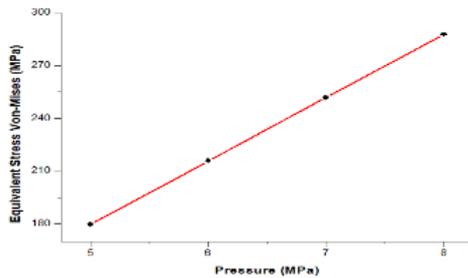


Fig. 2(a) Equivalent (Von-Mises) Stress

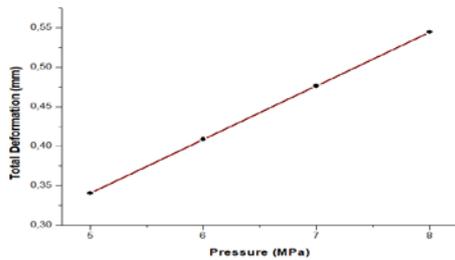


Fig. 2(b) Total Déformation

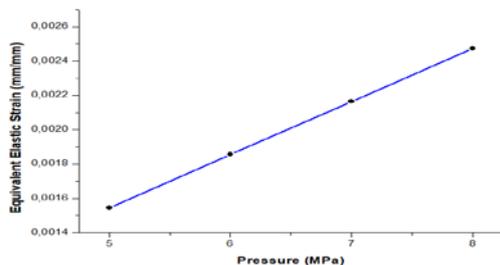


Fig. 2(c) Equivalent Elastic Strain

It is observed that Von Mises stress varies proportionately to internal pressure. Furthermore, the pipeline is in the elastic domain since the Von Mises stress at 8MPa pressure is less than the elastic limit. Deformations range from 0.34 mm (lowest value) to 0.54 mm (highest value).

The corresponding elastic stresses have very small values ranging from 0.0015 to 0.0024, making them virtually insignificant.

**CRACKED PIPE**

On the pipeline, we generated a semi-elliptical external fracture, the parameters and features of which are shown in the table below:

Table2. Information on the semi-elliptical crack

Definition	
Crack Shape	Semi-Elliptical
Major Radius	5. mm
Minor Radius	3. mm
Mesh Method	Tetrahedrons
Largest Contour Radius	1. mm
Element Size	Default (0.10071 mm)
Mesh Contours	6
Solution Contours	Match Mesh Contours

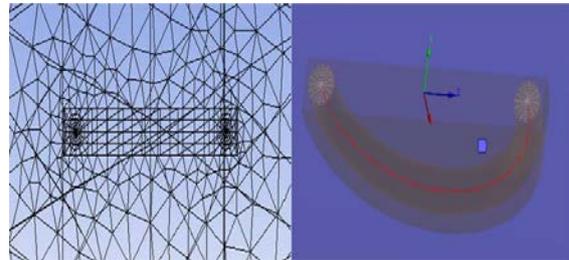


Fig. 3 Tetrahedral mesh and contours presentation (Zitouni et Labeled, 2021)

We used the same pressures that were tested in the same geometry tube. The goal of this method is to determine the impact of the highest pressure that our fractured pipe can resist. The Von Mises stress and the stress intensity factor are two strength criteria.

The results of this test are represented in the figures below.

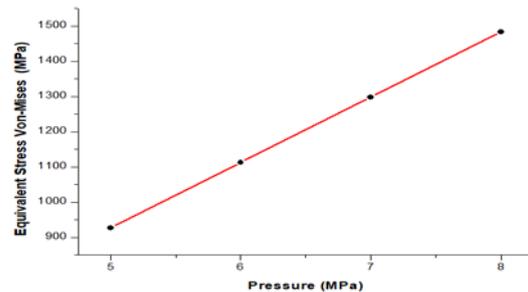


Fig. 4(a) Equivalent (Von-Mises) Stress

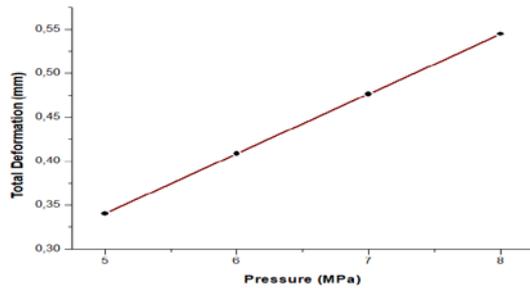


Fig. 4(b) Total Deformation

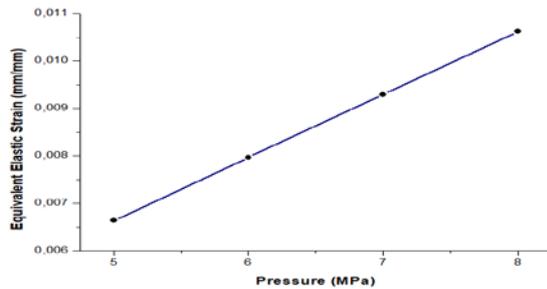


Fig. 4(c) Equivalent Elastic Strain

With the presence of the fracture and following Fig. 4 (a), we can see that the Von Mises stress has grown significantly and is now larger than the elastic limit for the lowest pressure. We may assume that the pressure of our fractured pipe can bear is less than 5 MPa. Beyond this value, the pipeline is expected to be damaged or to break suddenly.

### Stress intensity factor

George R. Irwin, widely recognized as the father of fracture mechanics, invented the stress intensity factor in 1957 (Irwin, 1957). It is one of the most basic and relevant factors in all fracture mechanics, and it is the sole essential parameter for determining the state of stress and strain in a crack (Fiordalisi, 2014). The variable  $K$  represents the Stress Intensity Factor (SIF). The stress intensity factor characterizes the stress condition near the fracture tip. It is proportional to the rate of crack development and is used to determine fracture failure criteria (Irwin, 195; S. Fiordalisi, 2014; Westergaard, 1939).

$K$  was defined by Irwin as a near approximation to the crack tip of Westergaard full solution for the stress field surrounding a fracture (Westergaard, 1939). Westergaard employed complicated numbers and the Airy stress function. Erwin discovered that the combination of and accurately reflected the severity of the stress condition at the crack tip, and while he did not include the value at first, it was subsequently added to get the following formula:

$$K = \sigma_\infty \sqrt{\pi a} \quad (1).$$

The figures below represent the stress intensity factors  $K_I$  and  $K_{II}$  from our study of the six contours for the four applied pressures (5MPa, 6MPa, 7MPa and 8MPa).

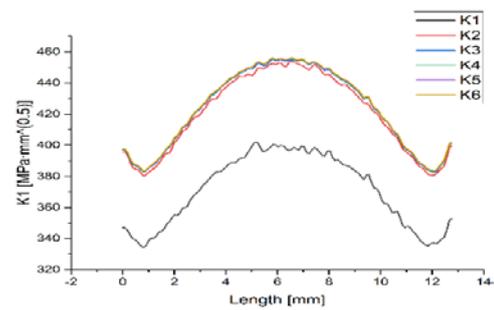


Fig. 5(a)  $K_I$  P = 5 MPa

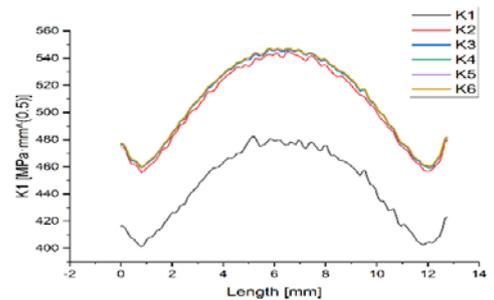


Fig. 5(b)  $K_I$  P = 6 MPa

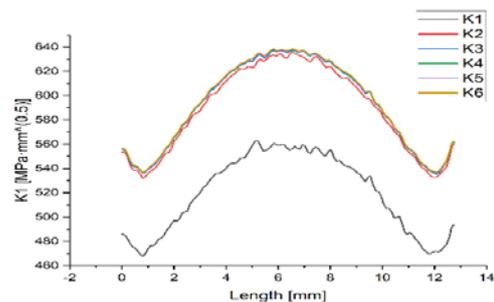


Fig. 5(c)  $K_I$  P = 7 MPa

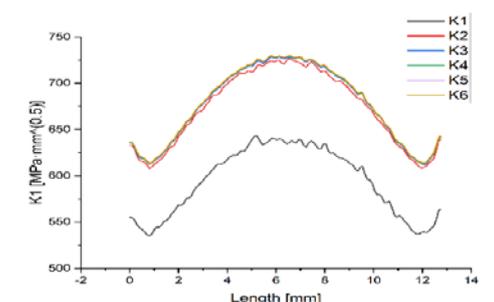


Fig. 5(d)  $K_I$  P = 8 MPa

With the exception of the first contour  $K_I$ , which has a lower value in contrast to the other contours, the contour curves are symmetrical with regard to the length of the semi-elliptical crack. Plasticization occurs as the applied internal pressure rises due to sudden cracking.

The contours of the stress intensity factor in mode II are then depicted in the figures below. For each pressure variation, the greater value corresponds to the first contour in mode II, as opposed to

the SIF in mode I. In Mode II, the SIF values for all five contours are nearly identical.

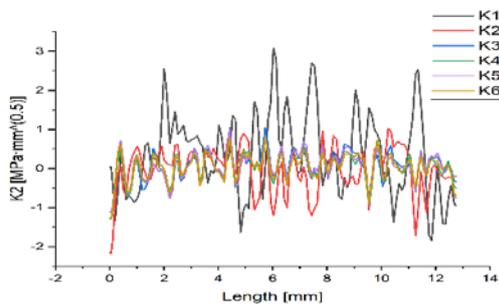


Fig. 6(a)  $K_{II}$  P = 5 MPa

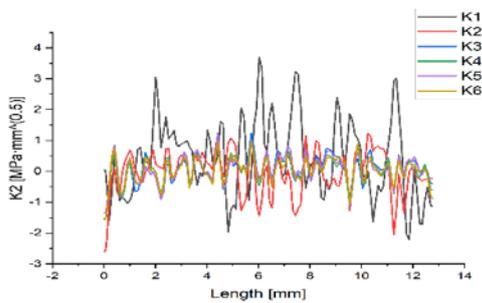


Fig. 6(b)  $K_{II}$  P = 6 MPa

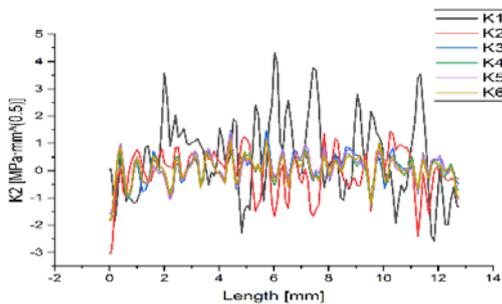


Fig. 6(c)  $K_{II}$  P = 7 MPa

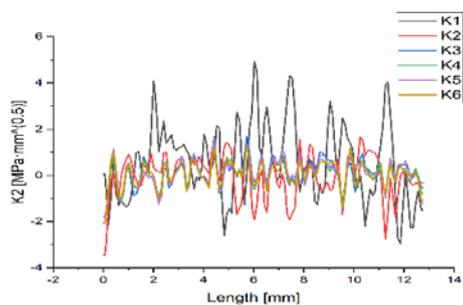


Fig. 6(d)  $K_{II}$  P = 8 MPa

Based on the findings obtained for different pressures (Fig. 5), it can be inferred that repair is required in the event of minimal pressure. We will suggest two repair methods, and the results will allow us to select the most successful repair method.

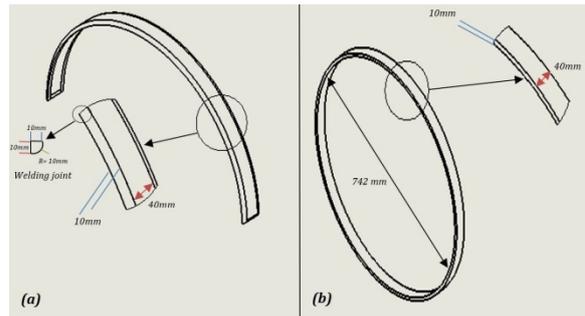


Fig. 7 Geometry of the two repair methods, (a). Repair of Half-Shell; (b). Repair by Composite Patch.

### REPAIR OF HALF-SHELL

This repair procedure entails inserting welded half-shells to manage the pipes that transport hydrocarbons in order to minimize potential leaks, and is therefore used to repair axial or circumferential through or non-through defects (Fodil et Aous, 2018). There are experimental works of this type of repair like the article of (Chapetti, M., Otegui, J., Manfredi, C., and Martins, C., 2001)



Fig. 8 Half-shell welding operation (A. Benlekhal, 2010)

In the third section of our work, we made a repair using a half-shell of the same X80M material as the pipeline, welded with an AA R610 weld bead, the mechanical characteristics are displayed in the table below:

Table3. Mechanical properties of AA R610 (Huissen., CEWELD, 2018)

Properties	MPa
Yield strength (YS)	620
Tensile strength (TS)	830
Young's Modulus	200000 MPa
Poisson's Ratio	0.3
Bulk Modulus	166670 MPa
Shear Modulus	76923 MPa

Fig. 9 represent the results of the simulation:

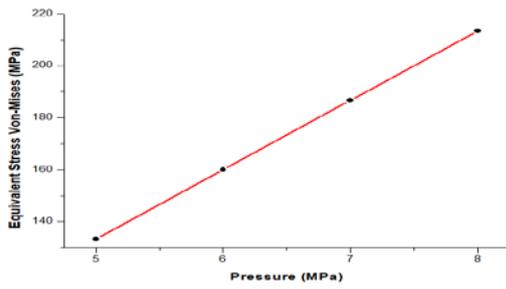


Fig. 9(a) Equivalent (Von-Mises) Stress

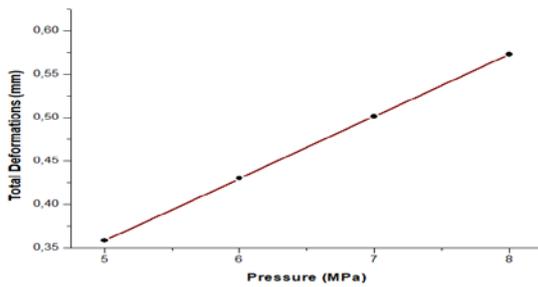


Fig. 9(b) Total Deformation

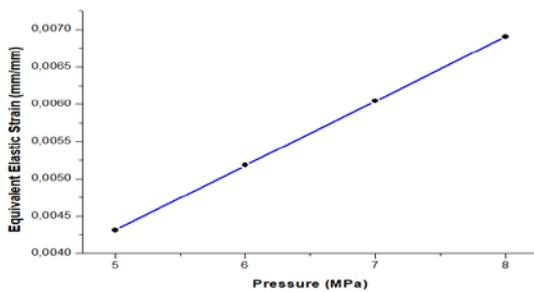


Fig. 9(c) Equivalent Elastic Strain

Fig. 9 show the effect of half-shell repair giving the pipeline extra life. We discovered that after the half-shell repair, the Von-Mises stresses were reduced by 86 percent when compared to the findings of the pipeline without repair.

By comparing Fig 2 (b) and Fig 9 (b) the deformation curves and the results are almost the same in both situations between 0.35 mm and 0.60 mm after applying the same 4 different pressures (without repair and with repair).

The  $K_I$  are shown in the graphs below:

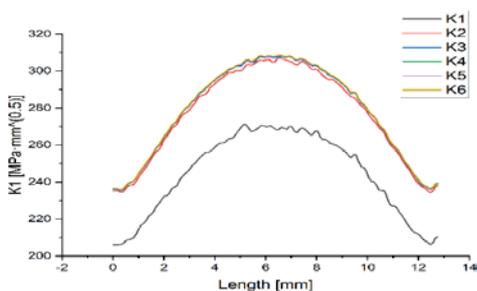


Fig. 10(a)  $K_I$  P = 5 MPa

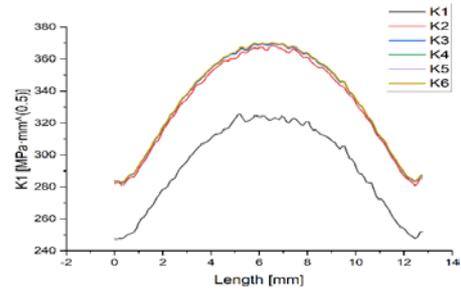


Fig. 10(b)  $K_I$  P = 6 MPa

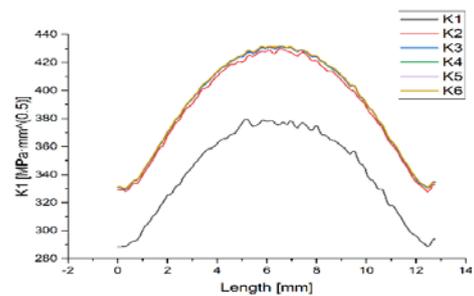


Fig. 10(c)  $K_I$  P = 7 MPa

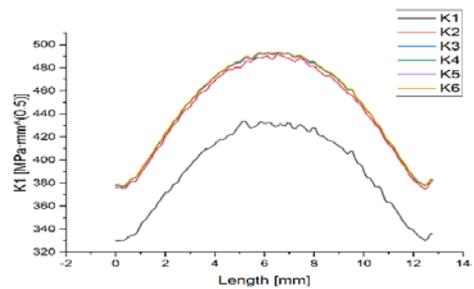


Fig. 10(d)  $K_I$  P = 8 MPa

The shape of the curves in this case with the half-shell repair is almost identical to the shape of the curves in the case of a cracked pipeline, the difference being that the values of the stress intensity factors  $K_I$  decrease almost by 66 percent when we apply pressure of 5 MPa, and the percentage decrease of the values of the factors becomes 40 percent with the other three pressures. It is noticed that the distribution of the curves is the same, with a significant drop in the level of Von Mises equivalent stresses.

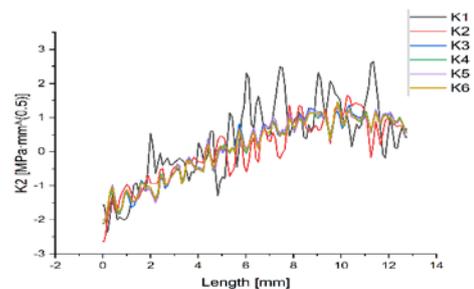


Fig. 11(a)  $K_{II}$  P = 5 MPa

We selected epoxy carbon for the composite patch repair, and its properties are as follows:

Table 4. Mechanical properties of epoxy carbon (Abdelouahed et Benzaama, 2019)

Young's Modulus X direction (MPa)	Young's Modulus Y direction (MPa)	Young's Modulus Z direction (MPa)
1.34e+005	10300	10300
Poisson's Ratio XY	Poisson's Ratio YZ	Poisson's Ratio XZ
0.33	0.33	0.53
Shear Modulus XY (MPa)	Shear Modulus YZ (MPa)	Shear Modulus XZ (MPa)
5500	5500	3200

The reparation white composite patch (epoxy/carbon) it is a popular method like the half-shell; there are also many experimental works of its own, such as (Duell, J.M., Wilson, J.M., and Kessler, M.R., 2008).

After the simulation we have the following results:

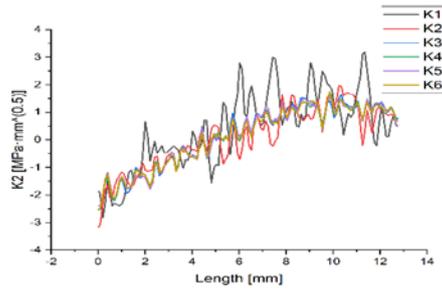


Fig. 11(b)  $K_{II}$  P = 6 MPa

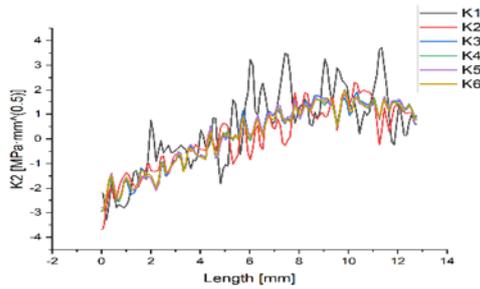


Fig. 11(c)  $K_{II}$  P = 7 MPa

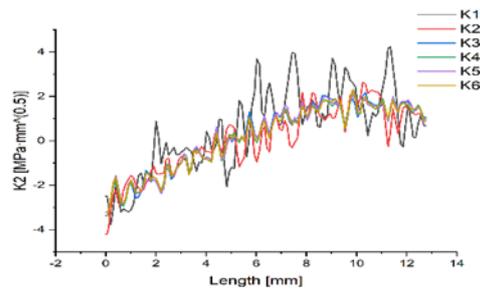


Fig. 11(d)  $K_{II}$  P = 8 MPa

### REPAIR BY COMPOSITE PATCH

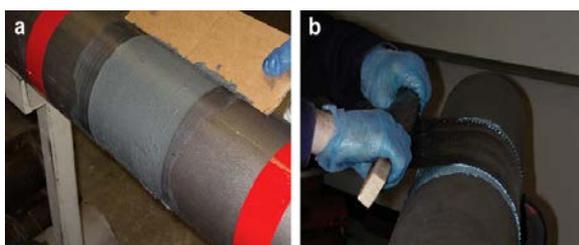


Fig. 12. Pipe repair process (a) filling defect with putty (b) wrapping epoxy wetted carbon fabric around the defect (Duell et Wilson, 2008).

This technique of repair is part of the damaged structure, particularly pipeline maintenance. New repair methods based on composite materials are being developed to slow the spread of cracks and increase the life of these structures. Indeed, because to their high specific mechanical characteristics and variety, composite materials and adhesives are increasingly being utilized to repair structures, with applications peaking around the end of the 1970s (Baghdadi, 2021).

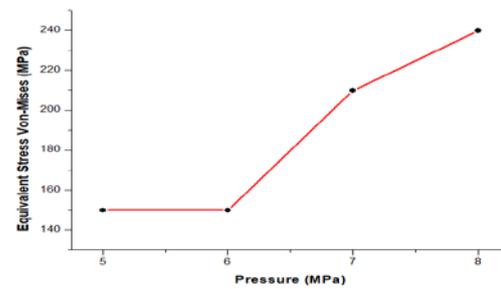


Fig. 13(a) Equivalent (Von-Mises) Stress

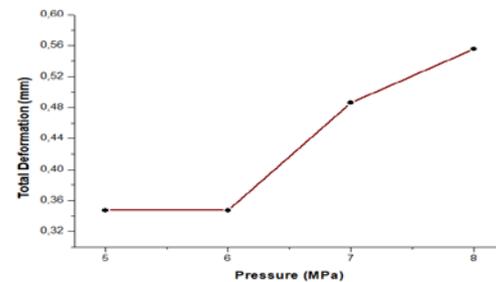


Fig. 13(b) Total Deformation

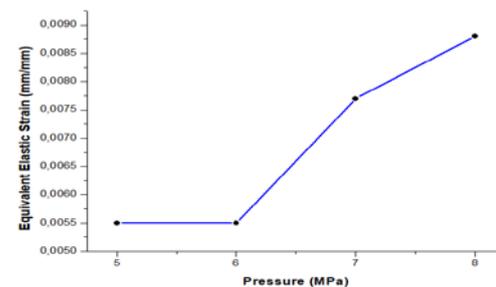


Fig. 13(c) Equivalent Elastic Strain

After the patch repair we have this time an almost 84% reduction in Von-Mises stresses, and almost 8% compared to the half-shell repair. In

addition, the Von Mises stress is constant for the two pressures 5 MPa and 6 MPa.

The  $K_I$  and  $K_{II}$  are on the following curves:

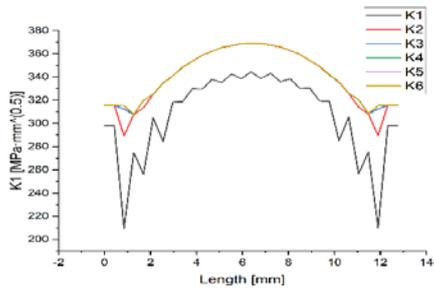


Fig. 14(a)  $K_I$  P = 5 MPa

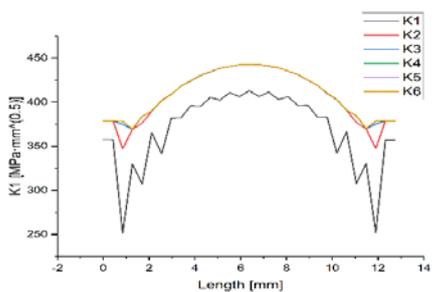


Fig. 14(b)  $K_I$  P = 6 MPa

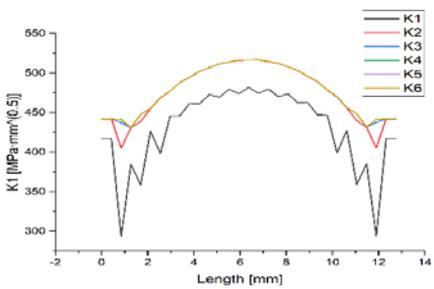


Fig. 14(c)  $K_I$  P = 7 MPa

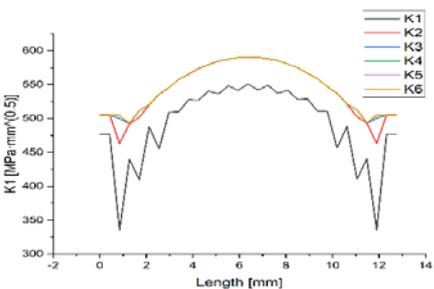


Fig. 14(d)  $K_I$  P = 8 MPa

The contour of  $K_I$  curves in the Patch repair is completely different from the other curves in the two prior examples, and we also observe that the four curves are symmetrical.

Compare the maximum values of  $K_I$  in this case with the case of repair by half-shell; we have an increase of almost 12% for the pressure 5 MPa, 16% for 6 and 8 MPa and 22% for 7 MPa.

The most remarkable thing in the results of Fig.

10 for the values of the SIF is the symmetrical appearance for the four pressures applied.

The comparison between the  $K_I$  of this repair with the  $K_I$  of cracked pipeline, we have a decrease of 19% for the three pressures 5, 6, 8 MPa and of 9% for 7 MPa.

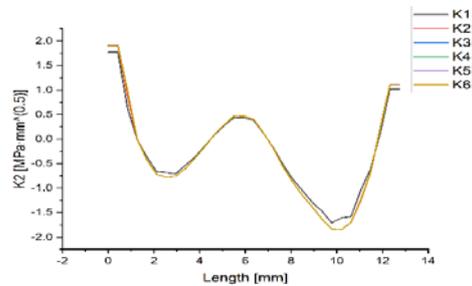


Fig. 15(a)  $K_{II}$  P = 5 MPa

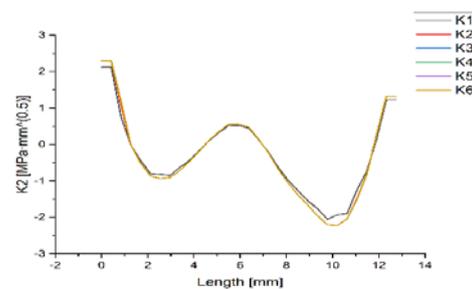


Fig. 15(b)  $K_{II}$  P = 6 MPa

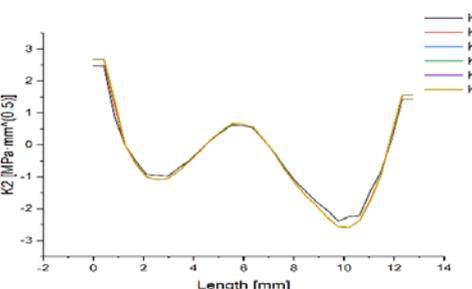


Fig. 15(c)  $K_{II}$  P = 7 MPa

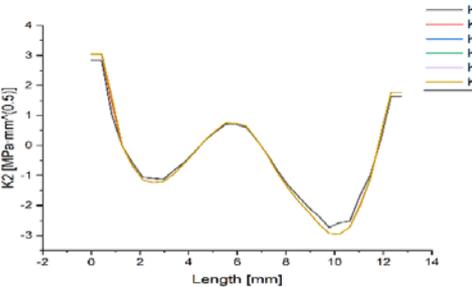


Fig. 15(d)  $K_{II}$  P = 8 MPa

We can see that the values of  $K_{II}$  are decreasing after Patch's repair.

We can say that all the contours of the SIF in mode II overlap, we can explain this by the fact that the whole cracking zone is well protected by this type of repair.

## CONCLUSIONS

We wanted to know the resistance of the pipeline to pressure variations in terms of the propagation of the initiated crack, on the one hand, and the interest in a repair, either patch or half shell, on the other. Fig. 16 shows the applications carried out on the pipeline while under internal pressure.

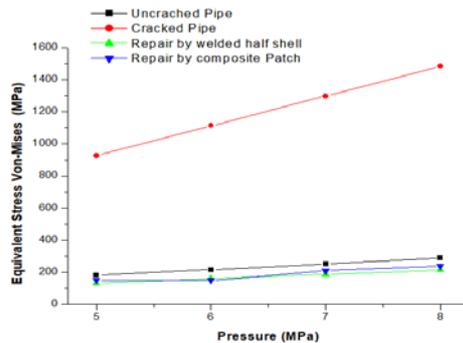


Fig. 16 Von Mises stresses for the four cases studied

### Before repair

- The pipeline operates and resists the influence of the internal pressures applied under normal circumstances (5, 6.7 and 8 MPa).
- Because of the presence of the initiated external semi-elliptical crack, we were able to see that the tube has a rather extensive plasticized zone in its proximity. Von Mises stresses rise in direct proportion to the amount of pressure applied.
- The contours of the stress intensity factor produced in mode I demonstrate the influence of the semi-elliptical fracture and the high level at each pressure variation. The first contour just indicates an initiation, but the other five contours clearly show a steady propagation of the fracture, as indicated by the superposition of the curves produced.
- In mode II, the SIF contours increase along the crack until canceling out at the crack tip.
- The value of the stress intensity factor in mode II is very low compared to  $K_I$ .

### After repair

We were able to stop the fracture from spreading by repairing it with a half-shell or a patch. Fig. 9 (a) and 13 (a) demonstrate the decrease in the extremely high level of Von Mises stress.

The effect of the presence of the crack on the strains is negligible.

- In opening mode (mode I), we can see that the five contours overlap with a symmetrical distribution and that the level of the SIF has decreased significantly.
- In shear mode (mode II), however, the value of the SIF tends to cancel out at the crack's end.
- Half-shell welding repair and composite patch repair (epoxy carbon) are excellent ways for

eliminating the damage of semi-elliptical cracks; these repair methods improve pipeline strength for a significant duration extra life.

- Half-shell repair strengthens the pipeline considerably more than composite patch repair, as evidenced by Von Mises stress values (difference of about 20 MPa).

The SIF is related by the crack, which means that the increase and reduction of SIF values are correlated with the presence of the fracture and its geometry (without forgetting the effect of the internal pressure). With fracture propagation, the findings of  $K_I$  rise and decrease (when the crack is reaching the average length the values decrease with the same increase values).

Both forms of repair, particularly repair by the half-shell, have an important influence in the drop in SIF (they have a significant decrease in the values of  $K_I$  and  $K_2$ ).

## REFERENCES

- Abdelouahed, E., Benzaama, H., Aour B., Mokhtari, M., and Tadjeddine, A.A., "Réparation par patch composite des pipelines sous charge de la pression interne," *24ème Congrès Français de Mécanique*. (2019).
- Arav, M., and Sanctuary, C., "Maintenance préventive: protection contre la corrosion, cas des grands chantiers," *41 Journées Techniques, Cefracor, Paris, France, (25-26 novembre 1997)*.
- Baghdadi, M., "Analyse paramétrique de la réparation par patch composites," *University Djillali Liabes De Sidi Bel Abbes* (2021).
- Benlekhal, A., "Rapport sur les procédures de réparation en charge de canalisation de transport d'hydrocarbures par demi-coquilles soudées," *SONATRACH-DRC, Oran-Arzew* (15 Août 2010).
- Brekke, H., "Design and material quality for high head turbine," *IAHR symposium, Montreal* (1986).
- Chapetti, M., Otegui, J., Manfredi, C., and Martins, C., "Full scale experimental analysis of stress states in sleeve repairs of gas pipelines," *International Journal of Pressure Vessels and Piping*, 78(5), 379–387, 2001.
- Duell, J.M., Wilson, J.M., and Kessler, M.R., "Analysis of a carbon composite overwrap pipeline repair system," *International Journal of Pressure Vessels and Piping*, 85(11), pp. 782–788 (2008).
- Fiordalisi, S., "Modélisation tridimensionnelle de la fermeture induite par plasticité lors de la propagation d'une fissure de fatigue dans l'acier 304L," *Higher school of mechanics and*

- aerotechnics* (2014).
- Fodil, M., and Aous, H., "Modélisation de la fissuration dans un pipeline en acier après réparation par patch en composite," *University Saad Dahleb Blida 1* (2018).
- He, B., Han, P., Lu, C., and Bai, X., "Effect of soil particle size on the corrosion behavior of natural gas pipeline," *Engineering Failure Analysis*, Vol 58, pp. 19–30 (2015).
- Huissen, T.G., Material list AA R610, The ASME classification is carried out based on a weld metal according to ISO standards 6851 (NL). *CEWELD Certilas Nederland BV, Gloxinialaan 2*, (Janvier 2018).
- Irwin, G.R., "Analysis of Stresses and Strains Near the End of a Crack Traversing a Plate," *Journal of Applied Mechanics*, Vol. 24, pp. 361-364, (1957).
- Kaddour, B., Bouchouicha, B., Benguediab, M., and Slimane, A., "Modeling and optimization of a cracked pipeline under pressure by an interactive method: design of experiments," *International Journal on Interactive Design and Manufacturing (IJIDeM)*, Vol 12, pp. 409–419 (2018).
- Kadri, M., Sahli, A., and Sahli, S., "Fracture Parameters for Cracked Cylindrical Shells," *Journal of Solid Mechanics* Vol. 11, No. 1, pp. 91-104 (2019).
- Khireche, A., and Labeled, Z., "Effet Des Defaults De La Corrosion Sur Le Comportement D'un Pipeline," *International Seminar in Industrial Engineering and Applied Mathematics (ISIEAM)* (2018).
- Khireche, A., and Labeled, Z., "Numerical study of a cracked pipeline under internal pressure," *Archives of Mechanical Technology and Materials*, Vol 40, pp. 30-33 (2020).
- Kim, Y.J., Shim, D.J., Nikbin, K., Hwang, S.S., and Kim, J.S., "Finite element based plastic limit loads for cylinders with part-through surface cracks under combined loading," *Int. J. Pres. Ves. Pip.*, vol.80, pp.527-540 (2003).
- Lu, H., Wu, X., Ni, H., Azimi, M., Yane, X., and Niu, Y., "Stress analysis of urban gas pipeline repaired by inserted hose lining method," *Composites Part B: Engineering* 183 (2020).
- Mahdi, H., and Hasnaoui, D., "Inspection et contrôle des assemblages soudés par méthodes non destructives", *University M'sila* (2018).
- Makino, H., Kubo, T., Shiwaku, T., Endo, S., Inoue, T., Kawaguchi, Y., Matsumoto, Y., and Machida, S., "Prediction for Crack Propagation and Arrest of Shear Fracture in Ultra-high Pressure Natural Gas Pipelines," *ISIJ International*, Vol. 41, No. 4, pp. 381–388 (2001).
- Meriem-Benziane, M., Abdul-Wahab, S.A., Zahloul, H., Babaziane, B., Hadj-Meliani, M., and Pluvinage, G., "Finite element analysis of the integrity of an API X65 pipeline with a longitudinal crack repaired with single- and double-bonded composites," *Composites Part B*, Vol 77, pp. 431- 439 (2015).
- Miller, A.G., "Review of limit loads of structures containing defects," *Int. J. Pres. Ves. Pip.* vol.32, pp.197-327 (1988).
- Olamide, A., Bennecer, A., and Kaczmarczyk, S., "Finite Element Analysis of Fatigue in Offshore Pipelines with Internal and External Circumferential Cracks," *Applied Mechanics* 1(4), pp. 193-223 (2020).
- Qiao, Q., Cheng, G., Wu, W., Li, Y., Huang, H., and Wei, Z., "Failure analysis of corrosion at an inhomogeneous welded joint in a natural gas gathering pipeline considering the combined action of multiple factors," *Engineering Failure Analysis*. Vol 64, pp. 126-143 (2016).
- Santos, T.F.A., Hermenegildo, T.F.C., Afonso, C.R.M., Marinho, R.R., Paes, M.T.P., and Ramirez, A.J., Fracture toughness of ISO 3183 X80M (API 5L X80) steel friction stir welds, *Engineering Fracture Mechanics*, Volume 77, Issue 15, (2010).
- Shuai, Y., Shuai, J., and Zhang, X., "Experimental and numerical investigation of the strain response of a dented API 5L X52 pipeline subjected to continuously increasing internal pressure," *Journal of Natural Gas Science and Engineering* Vol 56, pp. 81–92 (2018).
- Westergaard, H.M., "Bearing Pressures and Cracks," *Journal of Applied Mechanics*, Vol. 6, pp. A49-53 (1939).
- Yu, W., Zhang, J., Wen, K., Huang, W., Min, Y., Li, Y., Yang, X., and Gong, J., "A novel methodology to update the reliability of the corroding natural gas pipeline by introducing the effects of failure data and corrective maintenance," *International Journal of Pressure Vessels and Piping* 169, pp. 48-56 (2019).
- Zedadma, C., and Khallaf, N., "Étude, Réparation et Maintenance des Canalisations de Gaz," *University Constantine 1* (2018).
- Zhao, Y., and Song, M., "Failure Analysis of a Natural Gas Pipeline," *Engineering Failure Analysis*. Vol 63, pp. 61-71 (2016).
- Zitouni, T.A., and Labeled, Z., "Numerical Study on Dimensions and Orientation Effect of Semi-Elliptical Cracks in PE100 Pipelines," *International Journal of Applied Mechanics and Engineering*, vol.26, no.3, pp.198-207. (2021).

## NOMENCLATURE

$K$  stress intensity factor

$K_I$  stress intensity factor in mode I

$K_{II}$  stress intensity factor in mode II

$\sigma_\infty$  the infinite stress

$a$  the length of the crack

$P$  pressure